Validation of a GPU-Accelerated LBM Based Numerical Ice Tank for a Voith Schneider Propulsion System

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Fluid Dynamics and -Ship Theory

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- Introduction
 - Ice Simulations using LBM
 - VSP Kinematics
- Rainflow-Counting Algorithm / Fatigue Analysis
- Generic Test Case
- Real Application Test Case
- Summary and Outlook

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Introduction

Fluid Dvnamics

Ship Theor



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Lattice Boltzmann Method – Mesoscopic Method



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2/24

VSP Blade Geometry and Coordinate Systems



VSP Kinematics





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Data Postprocessing



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Linear Damage Accumulation / Fatigue Analysis



- Stress-Cycle Curve rates the performance of a component (depending on material and geometric properties, manufacturing processes etc.)
- LDA to assess the influence of a load spectrum on the durability of a component (match e.g. DNV GL class rules proof of operational strength)

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7/24

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Test Case Introduction

Single and **double** VSP propelled hulls analyzed in model tests and simulations



Generic *Feeding* Hull Geometry (model scale)

- Aims at maximizing the amount of ice floes approaching the propulsor
- SI dimensions: $5.7 \times 1.3 \times 1.25 \ m^3$ (L × B × H)

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Generic Ice Floe Geometry (model scale)

- Derived from the characteristic breaking pattern of the underlying hull geometry
- SI dimensions vary between:
 - edge length: $\mathbf{s_{Ice}} \in \{0.175, 0.35\}$ m
 - ice thickness: $\mathbf{h_{Ice}} \in \{0.05, 0.075, 0.1\}\mathrm{m}$

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Generic Feeding Hull – Domain Discretization



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Model Test vs. Simulation Animation



Homogeneous ice floe inflow approaching the propulsor

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- Partial *Milling* in model test (splitting up pre-brokes floes to [*very*] small pieces)
- Fairly good agreement between model test and sim. with respect to floe dynamics

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10/24

SimNo.	n [Hz]	$v_{\infty} [m/s]$	J [-]	$h_{Ice}\left[\mathbf{m} ight]$	$s_{\rm Ice}\left[{ m m} ight]$	Remark
1	3.00	0.45	0.119	0.05	0.1750	HSVA Model Test Configuration (BL)
2	2.25	0.34	0.119	0.05	0.1750	HSVA Model Test Configuration
3	3.00	0.45	0.119	0.05	0.3500	HSVA Model Test Configuration
4	2.25	0.34	0.119	0.05	0.3500	HSVA Model Test Configuration
5	4.00	0.45	0.090	0.05	0.1750	Increased Rotational Frequency
6	3.50	0.45	0.102	0.05	0.1750	Increased Rotational Frequency
7	2.25	0.45	0.159	0.05	0.1750	Decreased Rotational Frequency
8	3.00	0.90	0.239	0.05	0.1750	Increased Advance Ratio ($v_{\infty} \uparrow$)
9	3.00	1.35	0.358	0.05	0.1750	Increased Advance Ratio ($v_{\infty} \uparrow$)
10	3.00	1.80	0.477	0.05	0.1750	Increased Advance Ratio ($v_{\infty} \uparrow$)
11	3.00	0.45	0.119	0.75	0.2625	Increased Ice Thickness + Edge Length
12	3.00	0.45	0.119	0.10	0.3500	Increased Ice Thickness + Edge Length

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11/24

Simulations under Ice Conditions



Open Water vs. Ice Conditions – HSVA



• Reasonable shift in Local Drag Force

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- Increased Amplitudes as added resistance due to Milling
- Contacts occur at the profiles leading egde along the chord axis

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Open Water vs. Ice Conditions – **TUHH**



Local Blade (ID: 1) Lift+Drag Forces (elbe Simulation – 30 Periods)



• Significant filter (moving average) influence

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- Applied to hydrodynamic signal (smooth numeric oszillations)
- Currently chosen to match model test sampling rate (0.65ms)

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15 / 24

Ice Conditions – 4 TUHH Simulation Setups (equiv. to HSVA)

Local Blade (ID: 1) Lift+Drag Forces (Ice Conditions - 30 Periods) Local Blade (ID: 1) Lift+Drag Forces (Ice Conditions - 30 Periods) $s_{Ice} = 0.175m$ = const= const10³ 10³ $\mathbf{s_{Ice}} \cdot 2$ $\cdot 1.33$ Force Amplitude F_a [N] Force Amplitude F_a [N] $\cdot 1.33$ s_{Ice} · 10² 10² $\mathbf{n} \cdot 1.33$ $_{\infty} \cdot 1.33$ 10¹ 10 — F_{II}: Sim.–No. 1 - F_{II}: Sim.-No. 1 ··· F_u: Sim.-No. 2 --- F.: Sim.-No. 3 – F_v: Sim.–No. 1 – F_v: Sim.–No. 1 F_v: Sim.-No. 2 F_v: Sim.-No. 3 10^{0} 10⁰ 10³ 10³ 10^{2} 10⁰ 10² 10^{0} 10¹ 10^{1} Load Cycles N [-] Load Cycles N [-] Local Blade (ID: 1) Lift+Drag Forces (Ice Conditions - 30 Periods) Local Blade (ID: 1) Lift+Drag Forces (Ice Conditions - 30 Periods) $\mathbf{s_{Ice}} = 0.350 \mathrm{m}$ = const.= constSIce ' 10 10³ Force Amplitude F_a [N] Force Amplitude F_a [N] 1.33 $\mathbf{n} \cdot 1.33$ +1.33 $\cdot 1.33$ 10² 10² $\mathbf{s_{Ice}} \cdot 2$ 10¹ 10^{1} - F.:: Sim.-No. 2 – F_{II}: Sim.–No. 3 -- F.:: Sim.-No. 4 -- F.:: Sim.-No. 4 – F_v: Sim.–No. 3 F_v: Sim.-No. 2 F_v: Sim.-No. 4 F.,: Sim.-No. 4 10⁰ 10⁰ 10³ 10¹ 10^{2} 10⁰ 10¹ 10^{2} 10³ 10⁰ Load Cycles N [-] Load Cycles N [-] Numerical Ice Tank Involving Voith Schneider Propulsion Fluid Dynamics anc Ship Theory

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• HSVA

- Plausible results with regard to *Milling* process

- Vanishing influence of ice floe edge length
- Mass influence might exist for varying ice thickness (not tested so far)
- TUHH
 - Plausible results with regard to the *Rigid Body* model
 - Amplitudes increase whilst increasing momentum by either varying the VSPs frequency or the ice floe mass (edge length)

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Ice Conditions – TUHH Parameter Study I

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Local Blade (ID: 1) Lift+Drag Forces (Ice Conditions - 30 Periods)

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Ice Conditions – TUHH Parameter Study II





Ice Conditions – **TUHH** Parameter Study III

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Local Blade (ID: 1) Lift+Drag Forces (Ice Conditions – 30 Periods)

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20/24

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Voith Water Tractor – HSVA Model Test





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Voith Water Tractor – **TUHH** Simulation



Model Test vs. Simulation Animation



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Summary and Outlook

• Summary

- Implemented VSP kinematics for single and double VSP arrangement
- Validated hydrodynamic signal based on RANS and model test data
- Applied Rainflow-Counting algorithm
 - Prepare data according to class rules (e.g. **DNV GL**) for the proof of operational strength

• Ongoing and future work

- Construct realistic load scenarios / spectra for the life cycle of a VSP propelled ship (prescribe open water / ice cond. ratio)
 - Run further simulations of extensive paramater studies (Variables: frequency, inflow velocity, ice conditions, take icebreaking into account, etc.)
- Define S-N curve for the underlying VSP
- Accomplish LDA and assess the VSPs operational strength

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Thank you for your attention!

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Appendix



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Validation Sequence

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- Data to be used for the Validation of elbe
 - HSVA Model Test data
 - Voith RANS simulation CFD data (used for VSP dimensioning)
- Validation Sequence:



- Open Water \rightarrow Hydrodynamic Validation (**wo** ice)
 - * \rightarrow Single operational point considered: 90° VSP Pitch Angle, J = 0.12 [n = 3Hz, $v_{\infty} = 0.45$ m/s]
 - * Compare LBM, RANS and Model Test data
- <u>*Ice Conditions*</u> Validation
 - * Compare LBM to Model Test data
 - for 4 operational points $(n, v_{\infty} \text{ and floe size variation})$
 - * Run LBM Parameter Study to derive trends

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Hydrodamic Validation of decisive Quantities



Deviations between RANS and LBM result

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Open Water Validation – Rainflow Analysis (Lift)



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Open Water Validation – Rainflow Analysis (Drag)



Local Blade (ID: 1) Drag Force F_v (Open Water Conditions – 10 Periods)

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Ice Conditions – Case No. 1 – HSVA vs. TUHH (Lift+Drag)





Ice Conditions – 4 HSVA Model Test Configurations

